AMENDMENTS TO THE SPECIFICATION

Page 2, lines 6-8 should read:

During shaving, the cartridge assemblies can pivot forward ~~(clockwise)~~ or ~~backward (counterclockwise)~~ rearward from the neutral position relative to the handle, and the blade package can thereby follow the contours of the skin surface during shaving.

Page 2, lines 15-17 should read:

The blade cartridges for both the Schick Xtreame III razor and the Gillette Sensor Excel safety razor, ~~rotate~~ pivot about a center point pivot; and the cartridges may pivot in either direction ~~be rotated bi-directionally~~ from [[their]] a neutral position.

Page 2, lines 22-25 should read:

The Gillette Mach III safety razor is an example of a razor that features a guard-bar pivot (pivoting takes place on an axis through the guard-bar as opposed to a center point pivot); with the cartridge being capable of pivotal movement in only one direction ~~of unidirectional rotation~~ from its neutral position.

Page 2, lines 27-30 should read:

With a guard-bar pivot one obtains a "safer" shave than with a center pivot arrangement since applied loads (e.g., pressing the razor against the skin) are on the guard-bar and [[NOT]] not the blades. The guard-bar also facilities stretching of the skin compared with a center pivot system, thereby promoting a safe close shave.

Page 5, lines 15-21 should read:

When the blade assembly is in the first position, application of loading to the blade assembly will cause the blade assembly and the pivot assembly to ~~rotate~~ pivot about the center line of the blade assembly. Upon the application of additional load, the blade assembly will be moved against the first biasing member and will move to a second position relative to the pivot assembly. When the blade assembly is in the second position, application of loading to the blade assembly will cause the blade assembly and the pivot assembly to ~~rotate~~ pivot about the guard-bar axis of the blade assembly.

Page 5, lines 23-25 should read:

According to one embodiment of the invention, the pivot assembly is ~~bidirectionally~~ pivotable in both directions relative to the pivot frame and is free to pivot approximately ±20° from a neutral or rest position.

Pages 5, lines 31-34 should read:

The pivotal coupling between the blade assembly and the pivot assembly is preferably effected with a pair of bosses on the pivot assembly and a mating pair of pivot pockets or sockets formed in [[on]] the blade assembly. The bosses are approximately 180° ~~semicircular~~ semi-cylindrical and the pockets [[are]] subtend an angle of approximately 225° thereby allowing the blade assembly

Page 6, lines 1-6 should read:

to pivot approximately 45° relative to the pivot assembly. The pivotal coupling between the pivot assembly and the pivot frame is preferably accomplished via a pair of shell

bearing which include female journals on the pivot assembly, which are engaged by [[a]] corresponding male journals on the pivot frame. The second biasing member is preferably embodied as a resilient cantilever member ~~bendable finger~~, which extends from a position between the male journals and engages a central portion of the pivot assembly.

Page 6, lines 8-11 should read:

In the bi-directional embodiment, the resilient cantilever member ~~bendable finger~~ engages a pair of ~~downwardly extending~~ inverted U-shaped members. In the uni-directional embodiment, the resilient cantilever member ~~bendable finger~~ has a cam follower, which engages a cam surface on one side of the pivot assembly.

Page 7, lines 33-34 should read:

Turning now to Figures 1-5, an exemplary shaving system or razor assembly according to the invention includes a blade assembly (or cartridge) 10, a pivot assembly 12, and a

Page 8, lines 1-5 should read:

pivot frame 14, 14'. The blade assembly 10 includes a cap 16, a guard-bar 18, and a plurality of blades 20, 22, 24 arranged between the cap and the guard-bar. The lower interior of the blade assembly 10 includes a pair of sockets 26, one of which can be seen in Figure 2. The sockets have a pair of stops, 28, 30 which are angularly spaced approximately 225° apart.

Page 8, lines 7-11 should read:

The pivot assembly 12 has a pair of bosses, 32, 34 which are dimensioned to engage the sockets 26. The bosses are approximately 180° semi-~~circular~~ cylindrical bosses. Thus, ~~rotation~~ pivotal movement from stop 28 to stop 30 is approximately 45°. Adjacent the bosses, the pivot assembly has a pair of female journals 36, 38. Centrally, the pivot assembly has a cam surface 40 and a pair of ~~downward extending~~ inverted U-shaped members 42, 44.

Page 8, lines 18-22 should read:

The pivot frame 14 has a pair of male journals 46, 48 which are located and dimensioned to engage the female journals 36, 38 of the pivot assembly 12 to form shell rocker bearings which support the pivot assembly 12 to pivot about a virtual axis X fixed relative to the pivot frame 14 and located above the pivot frame 12. The axis X comprises the pivot axis of the shaving system. A cantilevered biasing member 50 is located between the male journals. The biasing member 50 includes an axle 52 which is located and dimensioned to engage the ~~downward extending~~ inverted U-shaped members 42, 44 of the pivot assembly 12.

Page 8, lines 24-28 should read:

The pivot frame 14' has a pair of male journals 46',48' which are located and dimensioned to engage the female journals 36, 38 of the pivot assembly 12 (shell rocket bearings). A cantilevered biasing member 50' is located between the male journals. The biasing member 50' includes a cam follower 52', which is located and dimensioned to engage the cam surface 40 of the pivot assembly 12.

Page 9, lines 1-11 should read:

the pivot assembly 12 to ~~rotate~~ pivot about the center line "C" of the blade assembly. Upon the application of additional load sufficient to overcome the reactive force exerted by the spring 11, the blade assembly 10 will be moved against the spring 11 and will move to a second position relative to the pivot assembly 12. Figure 8 illustrates the second position. When the blade assembly is in the second position, application of loading to the blade assembly will cause the blade assembly and the pivot assembly to ~~rotate~~ pivot about the guard-bar axis "G". From the foregoing, those skilled in the art will appreciate that the pivot assembly 12 illustrated in Figures 6-8 is free to pivot relative to the pivot frame approximately ±20° from the position shown in Figure 7. Further, it will be appreciated that the blade assembly is free to pivot relative to the pivot assembly approximately 45° from the first position shown in Figure 7 to the second position shown in Figure 8.

Page 9, lines 21-26 should read:

As described above, the invention increases safety by shifting the pivot point from a center point pivot (i.e., at the center blade on the shave plane), to a guard-bar pivot on the shave plane as shaving forces increase. The cartridge ~~rotates~~ pivots relative to the pivot assembly shifting ~~going~~ from a center pivot to a guard-bar pivot. It is returned to its initial position by the spring between the cartridge or blade assembly and the pivot assembly. The blade assembly ~~cartridge~~ and pivot assembly also move as a unit relative to the pivot frame.

Page 9, line 33 insert:

Further considering the illustrated shaving system or razor assembly and its operation, the pivot frame 14, essentially comprises an extension of the razor handle (not shown), and cooperates with the pivot assembly 12 to form a shell bearing which supports the pivot assembly 12 for arcuate rocking movement on and relative to the pivot frame 14 about a virtual axis or system pivot axis located above both the pivot frame and the pivot assembly. This virtual axis, which comprises the pivot axis of the shaving system is substantially fixed relative to the pivot frame 14 and the razor handle (not shown). The system's pivot axis is shown in Fig. 7 and indicated by the letter X.

The blade assembly 10, which includes the three blades 20, 22 and 24, the cap 16, and the guard bar 18, is supported for limited pivotal movement about a fixed axis on the pivot assembly 12, the latter axis being defined by cooperation of the outwardly projecting semi-cylindrical bosses 32 and 34 carried by the pivot assembly 12 and received in the inwardly open sockets 26 formed in the blade assembly 10. Thus, the blade assembly 10 is supported for pivotal movement on the pivot assembly 12 about a fixed axis and through an angle of 45 degrees between the stop surfaces 28 and 30 on the sockets 26 and coengagable abutment surfaces on the bosses 32 and 34 between a first position of the blade assembly relatively to the pivot assembly, shown in Fig. 7, and a second position of the blade assembly relative to the pivot assembly, shown in Fig. 8. A biasing spring 11, which comprises the first biasing means, acts between the pivot assembly 12 and the blade assembly 10 to urge the blade assembly 10 toward and to its first position of Fig. 7.

The resilient cantilevered spring member 50 carried by the frame assembly 14 acts between the frame assembly 14 and the pivot assembly 12 to retain various moveable parts

of the razor in a static or rest position when the razor is not in use and also allows 20 degree pivotal movement of the razor assembly 10 in either direction of rotation from the rest position and about the axis C, the degree of movement being controlled by the aforementioned shell bearings which couple the pivot assembly to the pivot frame.

It should be noted that when the blade assembly 10 is in its first position (Fig. 7) the virtual or system axis X is located substantially within the shaving plane and coincident with the leading edge of the center blade 22 in the blade group 20-24. Light shaving force applied to the blade assembly 10 may cause pivotal movement of the blade assembly 10 in either direction of rotational movement about the system pivot axis X in response to changes in skin surface contour and skin surface irregularities encountered during a normal shaving stroke. Since the spring 11 is responsive to a greater applied shaving force than the cantilevered biasing member 50, the blade assembly 10 will remain in its first position while the applied shaving forces are light. However, upon application of a heavier shaving force, that is a force of sufficient magnitude to overcome the reactive force exerted by the biasing spring 11, the blade assembly 10 will commence moving in a clockwise direction from its first position of Fig. 7 toward and ultimately to its second position (Fig. 8) causing the leading edge of the center blade 22 to move out of coaxial alignment with the system pivot axis X and further causing the guard bar 18 to take a position coincident with the system pivot axis, the latter position of the guard bar being indicated at G in Fig. 8. Thus, the shaving system of the present invention is sensitive to applied shaving force and is adapted to automatically shift the blade assembly from a center blade pivot position to a guard bar pivot position to accommodate changes in the magnitude of applied shaving force during the normal shaving process.